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1501 Kutchuan House, Hong Kong.



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ALLOWED 4/5/83
COMMUNION OF AUSTRALIA
PATENTS ACT 1952 - 1973
APPLICATION FOR A PATENT

INSTRUCTIONS

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Full name(s) of
applicant(s).

late

STANLEY WILLIAM OTTO MENZEL

Australian citizen, Company Director,

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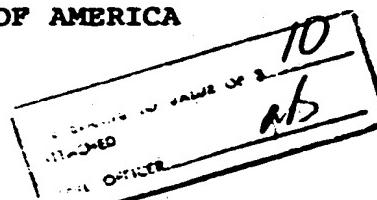
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AUSTRALIAN

6 OCT 1978

POST OFFICE

6814 Caminito Del Greco
SAN DIEGO
California
UNITED STATES OF AMERICA



Title of invention.

hereby apply for the grant of a Patent for an invention entitled

"MACHINE AND METHOD FOR FORMING TUBES FROM A STRIP"

which is described in the accompanying provisional/complete specification.

Cross out whatever
is not applicable

My/Our address for service is care of COLLISON & CO., Patent Attorneys,

Savings Bank Building, 97 King William Street, Adelaide, South Australia, 5000.

Dated this 5th day of OCTOBER 19 78

STANLEY WILLIAM OTTO MENZEL
By his Patent Attorneys
COLLISON & CO.

J C SCHMIDT

COMMONWEALTH OF AUSTRALIA
Patents Act 1952

**DECLARATION IN SUPPORT OF AN APPLICATION FOR A PATENT OR
PATENT OF ADDITION**

INSTRUCTIONS

In support of the Application made by

(a) **STANLEY WILLIAM OTTO MENZEL**

(b) Insert "of addition" if applicable

(hereinafter called "applicant(s)" for a patent

for an invention entitled

(c) Insert TITLE of invention

(c) **"MACHINE AND METHOD FOR FORMING TUBES FROM A STRIP"**

(d) Insert FULL name and address(es) of declarant(s) (See Note 1)

I/**We** (d) STANLEY WILLIAM OTTO MENZEL
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United States of America

do solemnly and sincerely declare as follows:

1. I am/**We** are the applicant(s)

(or in the case of an application by a body corporate)

1. I am/**We** are authorized to make this declaration on behalf of the applicant(s).

2. I am/**We** are the actual inventor(s) of the invention.

(or where the applicant(s) is/are not the actual inventor(s))

2. (e) I, the said STANLEY WILLIAM OTTO MENZEL, together with GILBERT WILLIAM VANCE and DAVID EARL MOMINEE, respectively of 6814 Caminito Del Greco, San Diego, California, USA, 3133 Vista Diego, Jamul, California, USA, and 6689 Japatul Valley Road, Alpine, California, USA

are the actual inventor(s) of the invention and the facts upon which the applicant(s) is/**am** entitled to make the application are as follows:—

(f) The said GILBERT WILLIAM VANCE and DAVID EARL MOMINEE have assigned the whole of their right title and interest in and to the said invention to me, the said STANLEY WILLIAM OTTO MENZEL, for Australia and all countries other than the United States of America

Declared at (g) San Diego

Dated (h)

Oct. 1, 1978

Stanley William Otto Menzel
Gilbert William Vance

NOTE:
No legal opinion or
other witness
required

PATENT, TRADE MARKS & DESIGNS SUB-OFFICE
12 OCT 1979
KENT TOWN SOUTH AUSTRALIA

TO: The Commissioner of Patents

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(54) FORMING TUBES FROM STRIP (HELICALLY)

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(57) A machine is also claimed.

Claim

1. The method of forming tubes from strip which comprises:

- (a) feeding forward a strip having a complementary rib and groove spaced apart on the said strip to each extend longitudinally on the said strip and oppositely facing on the said strip,
- (b) guiding the said strip into a helical configuration with the said rib in register with the said groove after one revolution of the said helical configuration to overlap the edges of the said strip, and
- (c) pressing together the said overlapping edges of the said strip on a joining roller while maintaining a differential drive pressure in the direction of the said feed between that part of the said strip being moved to the said joining roller and the tube formed by the helically wound strip beyond the said joining roller.

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Name of Applicant : TO BE COMPLETED BY APPLICANT
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Complete Specification for the invention entitled:

"MACHINE AND METHOD FOR FORMING TUBES
FROM A STRIP"

The following statement is a full description of this invention, including the best method of performing it known to me. us.

PATENT, TRADE MARKS
& DESIGNS SUB-OFFICE

= 29 OCT 1979

KENT TOWN
SOUTH AUSTRALIA

FIELD OF THE INVENTION

This invention relates to a machine for forming tubes from strip and in particular it relates to a machine of the type into which a strip of a selected width is fed and is 5. then curved around in a helical fashion and the edges of the strip are joined together by any suitable means, this operation continuing until the required length of tube has been achieved.

THE PRIOR ART

10. It is known to form tubes in this way and reference may be had to United States Letters Patent No. 3,199,541 of C.G. Richitelli in which one edge of the strip has a socket formation and the other edge of the strip has a bead formation so that when the strip is folded back on itself 15. in a helical fashion the bead can be pushed into the socket to join the two edges together and thus form a continuous tube.

It is also known to provide machines for effecting

the helical winding and as an example of such a machine we refer to United States Letters Patent No. 3,606,670 in the name of M. Wienand et al which uses a mandrel onto which the strip is held by means of rollers, the rollers being shaped to align and press together the two complementary edges of a strip when helically wound on such a mandrel.

Another such machine is shown in United States Letters Patent No. 3,938,558 in the name of Leroy Anderson, which machine uses a series of rollers which progressively

- 10. corrugates a strip of metal starting at the centre and gradually corrugating outwards and then winds the corrugated strip to helical form and forms a seam to join the contiguous edges to form a tube, the rollers which form the seam being disposed on both sides of the strip at
- 15. the point of contact of the strip as it is fed in.

A seam is first formed between contiguous edges of the strip and the strip is then curved to tubular form by further rollers positioned around the circumference of the tube.

- 20. A still further patent for forming metal strip into tubes is United States Letters Patent No. 3,621,884 in the name of John M. Trihey which again shapes a strip to a required corrugated form and then forms a seam between contiguous edges of the strip by shaping and pressing
- 25. together the edges to interlock.

OBJECTS OF THE INVENTION

It is an object of the invention to provide an improved method of tube formation in which a preformed strip, preferably extruded from a plastic, is helically wound and

- 30. the convolutions joined together to form a tube, a further object being to provide a close control of the way the strip

is bent and the convolutions interlocked to produce either a tube of uniform diameter or a tube of variable diameter along its length.

- It is also an object of the present invention to
5. provide certain improvements to machines of that type which form strip into tubular form by helically winding the strip.

It is a still further object to provide means whereby the strip, as the contiguous edges of helical convolutions of the strip are being joined, is given the required shape

10. and curvature.

It is a still further object to provide a machine in which the diameter of the helical tube being formed is variable.

- It is a still further object to closely control the
15. diameter of a tube formed in this way and to control this progressively as the tube is being formed so that the dimensions of the tube can be varied progressively in any required manner as the tube is being formed.

SUMMARY OF THE INVENTION

20. The method of forming the strip comprises feeding forward a strip having a complementary rib and groove spaced apart on the strip to each extend longitudinally on the strip and oppositely facing, and directing the strip into a helical configuration with the rib in register with the groove after
25. on convolution of the helix whereby to overlap the edges of the strip, and pressing together the overlapping parts of the strip on a joining roll while maintaining a differential feed pressure between that portion of the strip being fed in and that portion of the convolution of the
30. strip which is pressed against the part being fed in at the

joining roller.

- The machine comprises means to feed the strip forward over the joining roller and to curve the strip into a helical configuration beyond the joining roller and to join
5. the helical convolution of the strip to the feed strip at the joining roller by applying a differential feed pressure to the strip and the convolution at the joining roller.

- Thus the joining of the convolution of the strip to that part of the strip which is fed over the joining roller
10. while under differential pressure has the effect of urging the strip into a helical form to give a strong and accurately formed tube.

BRIEF DESCRIPTION OF THE DRAWINGS

- Figure 1 is a plan of a simple form of machine for
15. winding a strip into helical form.

Figure 2 is an end elevation of the machine illustrated in Figure 1.

Figure 3 is a detail of a modification of the illustrated machine.

20. Figure 4 is a cross section of a strip and joining roller which joins two edge portions of the strip when the strip is being wound to helical form, the section of the strip on the left being taken at the point of roller contact but the section of the strip on the right which is being fed
25. to form a helix being taken forwardly of the roller before the roller has positioned the strip in its interlocking configuration.

Figure 5 is a view corresponding to Figure 4 but

showing the edges of the strip interconnected.

Figure 6 is a section as on line 6 - 6 of Figure 5.

Figure 7 is a detail of a modified form of joining mechanism comprising a joining roller corresponding to the roller of Figure 6 but utilizing a driving roller to provide a differential drive for the strip and the first convolution of the tube whereby the required curvature of the strip is achieved to form a tube of a selected and variable diameter.

10. Figure 8 is a sectional view to reduced scale of the mechanism which controls the drive and the mechanism which regulates the dimension of the strip driving roller.

Figure 9 is a view corresponding to Figure 7 but using a divided roller to form a pair of driving rollers which 15. can be operated at differential rotation speeds to achieve the differential drive.

Figure 10 is a sectional view to a smaller scale showing the drive means for the rollers of Figure 9.

Figure 11 is a view showing a helical pipe being formed 20. by the mechanism illustrated in Figures 9 and 10 but showing means whereby the diameter of the tube can be progressively varied as the tube is produced.

Figure 12 is a detail of the adjusting mechanism of Figure 11.

25. Figure 13 is a plan of the machine shown in Figure 11, but showing the strip and tube in dotted lines.

Figure 14 is an end elevation of the machine shown in Figure 11 again showing the strip and tube in dotted lines.

Figure 15 is a somewhat schematic view of a machine with a similar function to that shown in part in Figures 5. 7 and 8 but showing how the feed and joining rollers can move circularly to allow the tube being formed to remain stationary.

DESCRIPTION OF THE PREFERRED EMBODIMENT

- Referring first to Figures 1 to 6 inclusive, the strip 10. 1 is fed forward by a roller 2 driven from a suitable power source such as a hydraulic or electric motor 3, the strip 1 being kept in contact with the roller 2 by the roller 4, at least one of the rollers being resilient to ensure a firm grip and drive for the strip.
15. The strip feeds from the rollers 2 and 4 to beneath a guide roller 5 which allows the strip to come into contact with the joining roller 6, from which joining roller 6 the strip passes on in contact with a series of guide rollers 7 positioned around the periphery of the tube which is being 20. formed by the strip, the rollers 5, 6 and 7 being mounted on spindles 8 held to a frame member 9 by nuts 10 on threaded shafts of the spindles, the shafts of the spindles passing through slots 11 in the frame member so that the rollers can be positioned to form tubes of different diameter.
25. Instead of using the slots 11 in the frame member 9 to vary the position of the spindles 8, each spindle 8, as shown in Figure 3, can be mounted on one end of an arm 14 connected by a pivot pin 15 to the frame member 9 and variable in angle by the other end of the arm 14 being connected by 30. a pin 16 to an adjusting member 17 having right and left hand screw thread portions 18 and 19, the one portion 19

engaging in the member 20 which engages the pin 16 while the other portion 18 engages in the member 21 which in turn is connected by a pin 22 to the frame member 9.

- In this way the position of the rollers 7 can be varied
5. to vary the size of the circle around which they are arranged, either individually or by a single adjustment that can move all rollers.

- It will be noted that the strip 1 is fed in at an angle to the axis of the tube being formed, see particularly
10. Figure 1, this being necessary to achieve the required helical formation. To ensure that the strip 1 is correctly positioned helically, the joining roller 6 and the series of rollers 7 each has on it a flange 25, which flanges are preferably movable along the rollers 6 and 7 to allow strips 15. of different width to be used, these flanges 25 engaging one edge of the strip to deflect the strip from the direction in which it is fed to the forming roller 6 to offset the edge so that the required helical formation results, and it will be noted that the series of rollers 7 are themselves 20. angled to correctly position the strip along its helical path. The flange 25 on the joining roller 6 is not always necessary.

- Referring now particularly to Figures 4, 5 and 6 it will be noted that the strip 1 has at its one edge portion a 25. socket 26 and at its other edge portion a rib 27 so that when the strip is helically wound the rib 27 coincides with the socket 26 at the point of contact with the forming roller 6, the strip also having a series of strengthening ribs 28 and a locking rib 29 according to a configuration for which 30. the machine is specifically adapted, but the strip can of course be of other form and need not be limited to the form shown.

- In operation the strip is fed forwardly by being gripped between the driving roller 2 and the pressure roller 4 and contacts the joining roller 6 as it is driven forward, the strip passing over the joining roller 6 to contact the
5. first of the series of rollers 7 and so on until the end of the strip, after one convolution, again reaches the forming roller 6 whereupon, because of the tension existing in the strip due to the bend which is imparted to it as it is helically curved, will cause this strip to be forced down
10. onto the roller 6 but, because it has been helically displaced, it will be forced down to cause the rib 27 to engage the socket 26 and by then continuing to feed the strip forward by means of the rollers 2 and 4 a helical tube is progressively formed.
15. The strip 1 is being forced forward by the rollers 2 and 4 against the resistance of the strip to be curved by the series of rollers 7 and the effect of this is that there is a differential feed pressure on the strip 1 as it is forced into joining contact with the convoluted strip and
20. this pressure so exerted has the effect of giving to the strip a curvature and lock required to form a successful helical tube.
- Because of the urging force of the strip 1 to rotate the tube itself a very effective seal is provided so that
25. the rib 27 is firmly engaged in the socket 26 under substantial pressure, and it will be seen for instance from particularly Figure 4 that the convolution B is forced down onto the joining roller 6 by the force applied as the strip is curved by the series of rollers 7 so that the strip 1 is
30. held very firmly down onto the joining roller 6, and it will

be seen also that the part A of the strip which is being fed in onto the roller 6 by the drive roller 2 and pressure roller 4 is pushed by the joining roller into the same plane as that part B of the strip already on the roller.

5. While in Figures 1 to 6 inclusive a form of the invention is shown in which the strip is forced onto the joining roller 6 by the curved configuration of the strip 1 as the forming takes place, it is possible to have a similar stress induced by the embodiment shown for instance in
10. Figures 7 and 8, in which dual joining rollers 30 are carried on a spindle 31 by bearings 32 but in this case instead of using the roller 2 which is the driving roller at a position remote from the joining roller 6, the driving roller, in this case numbered 33, is positioned to press onto the
15. forming dual rollers 30 the strip 1 as it is being processed into a tube.

As stated earlier there is a differential pressure between that part of the strip 1 as it is fed into union with that part of the strip which has already been formed into the helix, and to achieve this differential pressure the resilient roller 33 is mounted on a hollow shaft 34 which has a flange 35, to which shaft 34 and flange 35 the roller 33 is firmly bonded, the roller however being longitudinally compressible by means of a pressure member 36 on a shaft 37 which is movable in the hollow of a shaft 34 so that by moving the shaft 37 axially in relation to the shaft 34 that part of the roller between the flange 35 and the pressure member 36 can be varied in dimension, the roller being shown in that Figure with the pressure so applied that the part between the flange 35 and the pressure member 36 is of the same diameter as the part on the other side of the flange 35 but by for instance raising pressure the part between the flange and the pressure member can be reduced in diameter or by

increasing pressure it can be increased in diameter in relation to the part on the other side of the flange.

In this way by varying the diameter of that part of the roller 33 which is pressing down on the part B in the 5. illustration, a different forward rate of motion can be given to the Part B of the strip which is already part of a helix this causes the parts A and B to slip relative to each other in the joint between them. It has been found in practice that 10. this forms a very convenient way of selecting the size of the tube which is formed by helically winding the strip.

The method of varying the dimensions of the part of the roller 33 between the flange 35 and the pressure member 36 is shown particularly in Figure 8 and can consist of a thrust bearing 38 engaged by a fork 39 which has on it a threaded 15. stem 40 engaged by a hand wheel 41 which is axially confined between the members 42 of a frame member 43 which as will be seen is connected to a gearbox 44 having in it a worm wheel 45 driven from a worm 46 by a motor 47, the worm wheel 45 being secured to the shaft 34 so that the shaft 34 is both maintained 20. in correct axial position and is driven by the motor 47.

By setting the hand wheel 41 to give the required loading between the flange 35 and the pressure member 36 the required diameter of helical tube is produced.

In this form of the device the righthand portion of the 25. joining roller 30 is shown with its spindle 31 held to the flange 48 by a nut 49 to be adjustable in the slot 50, and the lefthand portion of the roller 30 can be similarly adjustably mounted. However, it will be realised that other forms of adjustment could be used to vary the distance between the 30. roller 33 and the joining roller 30 according to the dimensions of the strip being processed and the diameter of the roller 33.

In Figures 9 and 10 is shown a preferred form of device in which instead of using a single roller 33 with part of it

variable in diameter, separate resilient rollers 55 and 56 are used, the roller 55 being the main drive roller to feed the strip 1 into the helix while the roller 56 is the roller which controls the rate of revolution of the helical tube being formed, the roller 55 being mounted on a hollow shaft 57 while the roller 56 is mounted on a separate shaft 58.

In this case the hollow shaft 57 is connected to a worm wheel 60 which in turn engages the worm 61 which is driven by a hydraulic or electrical motor 62, the assembly being positioned in the housing 63 which forms a gearbox to drive the roller 55.

The shaft 58 is similarly connected to a worm wheel 68 engaging a worm 69 driven from a motor 70, the housing 71 being attached to a support member 72 to which the gearbox 63 is also attached, the gearbox 63 again having a bracket 73 which supports the shaft 74 of the two forming rollers 75 and 76, which have bearings 77 engaging the shaft 74. The shaft 74 is again positionable in a slot 78 in the bracket 73 so that by loosening the nut 79 the distance between the rollers 75 and 76 and the corresponding rollers 55 and 56 can be selected. Alternatively rollers 76 and 75 can be individually mounted and spring loaded.

In the case of this embodiment of course the curvature given to the strip as it is formed into a helix is adjusted by varying the speed of the motors 62 and 70 and it will be realised for instance that the rollers 56 can be driven at a different speed to the roller 55 by varying the speeds of the motors, and this then has the effect of varying the pressure on the part A of the strip 1 as it is fed forward between the rollers 55 and 75 in relation to the speed of that part B of the strip 1 between the rollers 56 and 76,

this differential pressure ensuring that while the two overlapping edges of the strip are forced together to engage the rib 27 in the socket 26 one of these parts of the strip is held back relative to the other part of the strip by the differential speed of the rollers 55 and 56 to cause the joined strip to be given a curvature due to the differential pressures existing in the two contiguous parts of the strip 1 being joined.

This again allows the diameter of the helical tube being formed to be varied simply by varying the differential speed of the motors 62 and 70 but the form of the invention now to be described with reference to Figures 11, 12, 13 and 14 adds a convenient form of control to the operation of the motors 62 and 70 and allows by a simple adjustment 15. of a hand wheel a variation of the diameters of the tube being formed.

Referring now to Figures 11, 12, 13 and 14 it will be seen that again the rollers 55 and 56 are used, driven by the motors 62 and 70 through the gearboxes 63 and 71, but the differential speed can be varied by varying the speed of one of the motors 62 or 70 by means of a sensing roller 85 which is mounted in a fork 86 on a stem 87 which in turn is movable in an arm 88 to actuate a transducer 89 according to the pressure being exerted on the roller 85 by the strip 1, the roller 85 moving outwardly if the diameter of the tube increases and inwardly if the diameter of the tube decreases. To allow this sensing to take place for different diameter tubes, or variable diameter tubes the arm 88 is mounted on a slider 90 on a frame member 91 which supports a threaded shaft 92 confined axially by bearings 93 and 94, the threaded shaft 92 having on it a bevel wheel 95 which meshes with a bevel wheel 96 which in turn is fixed to a shaft 97

having left and right hand threads on it again axially confined in bearings 98 and 99 and having on it a hand wheel 100 so that by rotating the shaft 97 by means of a hand wheel 100 the slider 90 can be moved up and down to fix the

5. position required to produce a tube of a certain diameter, but to avoid distortion of the tube as it is formed the shaft 97 actuates a pair of sliders 102 operating on lateral extensions 103 of the frame member 91 and carrying rollers 104 on brackets 105 so that rotation of the shaft

10. 97 through the bevel wheels 95 and 96 simultaneously moves the rollers 104 either inwards or outwards according to the required diameter, but the sensing roller 85 is moved up and down at a proportional rate to effect the required sensing.

15. By coupling the transducer 89 to control the speed of one of the motors 62 or 70 it is thus then possible to vary the differential speed to maintain the diameter of the spiral being formed through the sensing roller 85 which changed the speed of the motor according to the position of the roller

20. 85 in relation to the arm 88 which in turn actuates the transducer 89 to either increase or decrease the speed of the relevant motor to maintain the correct curvature of the strip 1.

The strip 1 actually is fed in by means of a guide 107

25. which is shaped to feed the strip correctly to the roller 55 and this guide has on it rollers or pins 108 which confine the strip correctly within the guide 107 but allowed to be drawn forward by the roller 55 at the correct speed to be forced into contact with the rollers 55 and 75.

30. In this way if for instance the diameter of the tube is to be increased the hand wheel 100 is simply rotated to move the arm 88 outwards and simultaneously to move the arms 105 outward but at half the rate of movement of the arm

- 88 and the changed pressure on the roller 85 will automatically compensate either the motor 62 or 70 to achieve this varied diameter of the helical tube. This assembly therefore makes it possible to produce a
5. progressively tapered tube, or a tube varying substantially in diameter along its length according to a required program.

In normal operation, in the production of constant diameter tubing, both the drive rollers 55 and 56 turn at approximately the same speed and preferably with the main

10. drive roller 55 turning at a constant speed. The roller 56 may be varied in speed under the control of the size sensing roller 85 to correct any change in diameter of the helical convolution being produced.

- Any increase in the helical diameter causes the size
15. sensing roller to rise from its normal position, which through electronic control means varies the speed of the motor 70 driving the roller 56, with the roller 56 then increasing in speed. This increase in speed of the roller 56 in relation to the roller 55 reduces the diameter of the
20. helix being produced until the size-sensing roller 85 has returned to its normal position.

If the size-sensing roller drops below its normal position, the roller 56 will reduce in speed until the helix has caused the sensing roller to return to its normal position.

25. To change the diameter of the helixal tube, the horizontal sensing rollers 104 are moved into position relative to the axial centre line of the helical tube. This change may be made at any time either before or during the time that the helix is being produced.
30. As the rollers 104 are connected to move with the roller

85 but through half the distance when the hand wheel 100 is rotated the sensing roller 85 will be positioned to vary the speed of the roller 56 and the diameter of the helical convolution which is being produced will change, thus

5. permitting the diameter of a tube to be progressively varied as it is produced either to produce a tube having a constant taper along its length or a tube of varying dimension along its length.

As the tube being formed rotates in any of the foregoing

10. embodiments support means are provided such as the frame bars 106 having on them anti-friction roller 109 which frame bars in turn may be carried by means of a "Y" shaped assembly mounted on a common base which in turn may be mounted on a mono-rail (not shown).

15. In the form shown in Figure 15 the joining roller 110, which corresponds to the roller 30 of Figure 7, and the feed roller 111, which corresponds to the compound feed roller 33 of Figure 7 are carried on a frame 112 which can rotate in a ring 113 on a chassis 114 provided with transport wheels 115, the frame 112 also carrying a motor 116 which drives the feed roller 111 through a chain 117. In this embodiment a strip 1 is fed between the rollers 110 and 111 and when a first convolution has been formed it is held against rotating in any suitable manner and if drive 20. continues the rollers 110 and 111 force the frame 112 around in the ring, and the chassis 114 is moved forward as the helical tube is formed.
25. continues the rollers 110 and 111 force the frame 112 around in the ring, and the chassis 114 is moved forward as the helical tube is formed.

To allow the rotation the frame 112 has on its

30. circumference a track 120 engaged by rollers 121 rotatably carried in the ring 113 fixed to the chassis 114.

As in operation the frame 112 rotates as the drive roller 111 drives the frame around by reaction of the helical

tube which is prevented from rotating. The strip 1 must be fed to between the rollers 110 and 111 and this can be achieved by mounting a spool 122 with a coil of strip 123 on it (shown in dotted lines) on a support shaft 124 on 5. the frame 112 the strip can be unwound and by following generally the path of the arrow 125 will feed to between the rollers 110 and 111.

The drive roller 111 is of similar construction to that shown in Figure 7, for the roller 33, that is it has a resilient portion X and a resilient but expandable portion Y which is tensioned by the nut 127 to determine the required differential drive to produce the required diameter of the helically formed tube.

This form of device is particularly suitable for large 5. tubes which, for instance, must be embedded in the ground, and can be formed by allowing the chassis 114 to move forward as the tube is manufactured.

This principle can, of course, be applied in various ways but the machine must be shaped to allow free feed to 0. the rollers 110 and 111.

The electrical feed to the motor 116 can be taken to it in any suitable manner.

The machine described in this Figure may be changed for hand operation by simply mounting the spindles of the 5. rollers 110 and 111 on an arm secured to a spindle on a support frame at the axis of the tube to be formed and rotating the roller 111 by a hand crank whereupon the rollers will drive the arm around the helical convolution of the tube.

Obviously the roller 111 can be replaced by the two-0. roller assembly shown particularly in Figure 9 rather than

the embodiments shown which uses a single variable-diameter roller.

- So far as the sealing of the helical joint in the tube is concerned, this can be achieved by engaging the rib 27
5. in the groove 26 and locking it by forcing the locking rib 29 behind the expanded end of an adjacent rib 28 of the strip 1, but a sealant can be applied to the join through, for instance, a tube T as shown in Figure 1. Sealant could be pre-applied to the strip or a sealing compound could be
10. used, and if the helical tube is to be strengthened a strengthening member such as a wire or a high tensile ribbon could be fed in with the strip, for instance, at the area designated 131 in Figures 4 and 5 or at any of the areas 132 between the ribs when these are present.

THE CLAIMS DEFINING THE INVENTION ARE AS FOLLOWS:

1. The method of forming tubes from strip which comprises:

- (a) feeding forward a strip having a complementary rib and groove spaced apart on the said strip to each extend longitudinally on the said strip and oppositely facing on the said strip,
- (b) guiding the said strip into a helical configuration with the said rib in register with the said groove after one revolution of the said helical configuration to overlap the edges of the said strip, and
- (c) pressing together the said overlapping edges of the said strip on a joining roller while maintaining a differential drive pressure in the direction of the said feed between that part of the said strip being moved to the said joining roller and the tube formed by the helically wound strip beyond the said joining roller.

2. The method of claim 1 wherein the said differential pressure is achieved by feeding the said strip to the said joining roller while guiding the said strip beyond the said joining roller by engaging the said strip on spaced rollers having projecting collars arranged around a circle and urging the said strip into helical configuration by the said rollers and collars to force the said overlapping edges together on the said joining roller.

3. The method of claim 1 wherein the said differential pressure is achieved by feeding the said strip to between the said joining roller and a pressure roller, driving one of the said rollers

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5 to move the strip forward longitudinally guiding
the said strip beyond the said rollers into a helical
convolution to bring the said rib in line with the
said groove at the said rollers, also engaging the
helical convolution of the strip between the said
rollers, and continuing to drive the said strip
forward while pressing the said rib into said groove
while maintaining the differential drive pressure
on the said strip and that part of the strip forming
the helical convolution to cause the said strip
10 when joined to the said convolution to ensure and
maintain a helical configuration by a stress
differential.

4. The method of claim 3 wherein the said strip
and the said convolution are passed between the
said rollers at a location where at least one said
roller has different diameters at the said strip
15 and the said convolution whereby to achieve the
differential drive pressure.

5. The method of claim 3 wherein the said strip
and the said convolution are passed between the
said rollers but one roller is divided to have two
sections with different adjacent diameters, one
to engage the said strip and the other the said
convolution, and driving the sections of the divided
roller at different rates of rotation whereby to
achieve the differential drive pressure.

6. The method of claim 5 wherein the speed
of at least one of the said sections of the said
divided roller is variable and varying the said
speed to produce a different diameter in the helical
tube being formed.

7. The method of claim 6 wherein the speed of the said variable speed roller is controlled by engaging a sensing member on the said helical tube, and causing the said speed to be varied by
5 the said sensing member.

8. The method of any one preceding claim wherein the rollers engaging the strip to provide the differential drive pressure to the strip and helical convolution are free to move circularly about the
5 axis of the tube being formed, which consists in holding the helical convolution stationary about its longitudinal axis and rotating the said rollers to drive them around the said longitudinal axis.

9. A machine for forming tubes from strip, which strip has a complementary rib and groove spaced apart on said strip to each extend longitudinally on the said strip and oppositely facing on said
5 strip, the said machine comprising:

(a) a joining roller rotationally supported on a support frame to engage the said strip on one side thereof,

(b) means to urge the said strip forward over the said joining roller, said joining roller being positioned to allow the said strip to be curved around into a helical convolution to again engage the said joining roller to overlap at least the edge portions of said strip and to align the said rib with the said groove,

(c) means to press the said overlapping edge portions together while maintaining a differential forward urging pressure on the two contiguous parts of the strip which are positioned on

the said joining roller whereby to force the said rib into the said groove to progressively form a helical tube from the said strip.

10. A machine according to claim 9 characterised by a series of guide rollers beyond the said joining roller arranged around a circle on spindles on the said support frame to deflect the said strip as it is urged forward into a helical convolution to overlap at least the edge portions of the said strip to align the said rib and groove and to force the convolution down on to the said joining roller to engage the said rib with the said groove, whereby the said differential forward urging pressure is achieved by urging the said strip forward against a retarding effect produced by the said guide rollers which engage the convolution of the strip.

11. A machine according to claim 10 further characterised by collars on the said guide rollers progressively displaced to guide the said strip into a helical path by engaging the edge of the said strip.

12. A machine according to either one of claims 9, 10 or 11 wherein the said joining roller and the said means to urge the said strip forward and the said guide rollers are mounted on the said support frame, and by means to allow the said support frame to rotate about the axis of the said tube being formed.

13. A machine according to claim 9 characterised by a further roller also carried on the said support frame positioned to engage the said strip on the remote side of the said strip to press the said strip on to the said joining roller, at least one

of the said rollers being driven to form a drive roller and at least one of the said rollers having a different diameter at the area where the said roller engages the said strip and where the convolution 10 engages the said roller, whereby the said rib is forced into the said groove by differential pressure applied to the said strip and the said convolution between the said rollers whereby to drive the said strip to curve the said strip to form and maintain 15 the helical convolution.

14. A machine according to claim 13 wherein the said roller which has a different diameter where it engages the said strip to where it engages the said convolution, has at least one of the said diameters 5 variable to change the diameter of the convolution produced thereby.

15. A machine according to claim 9 characterised by a further roller carried on the support frame engaging the said strip on the other side of the said strip to press the said strip on to the said 5 joining roller, at least one of the said rollers being driven and at least one of the said rollers being divided to have one part engage the said strip and the other part to engage the said convolution at the said joining roller, and further characterised 10 by means to cause the said two parts of the roller to have different peripheral speeds.

16. A machine according to claim 15 wherein the said divided roller is the driven roller and the two parts of the divided roller are driven to have different peripheral speeds.

17. A machine according to claim 16 wherein the said two parts of the driven roller are of similar diameter but are driven at different rates of rotation.

18. A machine for forming tubes from strip, which strip has a complementary rib and groove spaced apart on the said strip to each extend longitudinally on the said strip and oppositely facing on the said strip, the said machine comprising:

(a) means to apply pressure to the said strip to feed the said strip forward longitudinally over a joining roller carried on a support frame,

(b) means to curve the said strip into a helical configuration beyond the said joining roller to overlap at least opposite edge portions of the said strip and to align the rib of the said strip with the groove of the said strip on completion of one convolution of the said helix at the said joining roller, and

(c) means to engage the said rib in the said groove by differentially varying the feed pressure between that part of the said strip being fed over the said joining roller and the convolution of the said strip where it engages the said joining roller.

19. A machine according to claim 18 wherein the said feed means comprise a pair of feed rollers to engage the said strip and feed it forward longitudinally, over the said joining roller, and wherein the said strip is curved by a series of rollers also in the path of the said strip beyond the said joining roller arranged around a circle to guide the strip back to the said joining roller, and means to deflect the said strip to form a helical convolution

10 to overlap two edge portions of the said strip and
to join them by engaging a rib on one side of the
strip toward or at one edge thereof and a groove
on the other side of the said strip toward or at
the other edge thereof, whereby the said strip rotates
15 the said convolution and applies a differential
force between the said strip and the said convolution
at the junction to firmly interengage the said rib
and groove on the said joining roller.

20. A machine according to claim 19 wherein
the said means to deflect the strip comprise peripheral
flanges on the said series of rollers positioned
to engage the edge of the said strip to displace
5 the said strip laterally.

21. A machine for forming tubes from strip,
which strip has a complementary rib and groove spaced
apart on the said strip to each extend longitudinally
on the said strip and oppositely facing on the said
5 strip, the said machine comprising:

- (a) a joining roller arranged on a support frame to rotate,
- (b) means to guide a strip to feed over said joining roller,
- 10 (c) a first roller arranged to engage the said strip on the other side to the said joining roller to maintain the said strip in contact with the said joining roller, the said roller also being supported on the said frame,
- 15 (d) a second roller adjacent and axially aligned with the first roller arranged to engage a convolution of the said strip to maintain the said convolution in contact with the said joining roller,

20 tel means to drive the said first roller and the said second roller independently, whereby the said strip can be engaged between the first roller and the joining roller and curved around beyond the said rollers to form a helical convolution to feed the strip to overlap at least opposite edge portions of the said strip and align the rib of the said strip with the groove of the said strip on completion of one revolution, and the said rib can be forced into the said groove by pressure between 25 the said second roller and the said joining roller to maintain a differential pressure between the overlapping edge portions of the said strip which are being joined.

30 22. A machine according to claim 21 wherein the said first and the said second rollers are at least faced with a resilient rubber-like material.

5 23. A machine according to claim 21 or 20 wherein driving means for the said first and the said second rollers have the rate of rotation of at least one of the said rollers variable by mechanism connected to a sensing member in the path of a said helical convolution formed from the strip whereby to maintain a selected diameter of the said helical convolution by increasing or decreasing the differential speed of the said first and the said second roller according 10 to the pressure applied on the said sensing member.

5 24. A machine according to either claim 21, 22 or 23 wherein the said first roller is mounted coaxially with the said second roller on coaxial shafts supported in bearings on the said support frame, said frame also supporting a pair of guide

members positioned one toward each side of the said first and the said second rollers so as to engage opposite sides of a helical convolution formed by a strip urged forward by the said first roller,
10 and a further guide member disposed between the said pair of guide members but opposite the said first and second roller in relation to the said helical convolution, a speed control for at least one of the said driving means for the said first
15 and second rollers, means interconnecting the said further guide member with the said speed control, and means to move the said guide members simultaneously to define a larger or smaller circle passing through a plane between the said first and second roller and the said joining roller whereby to vary the rate of drive between the said first and the said second roller to vary the diameter of a tube being formed.
20

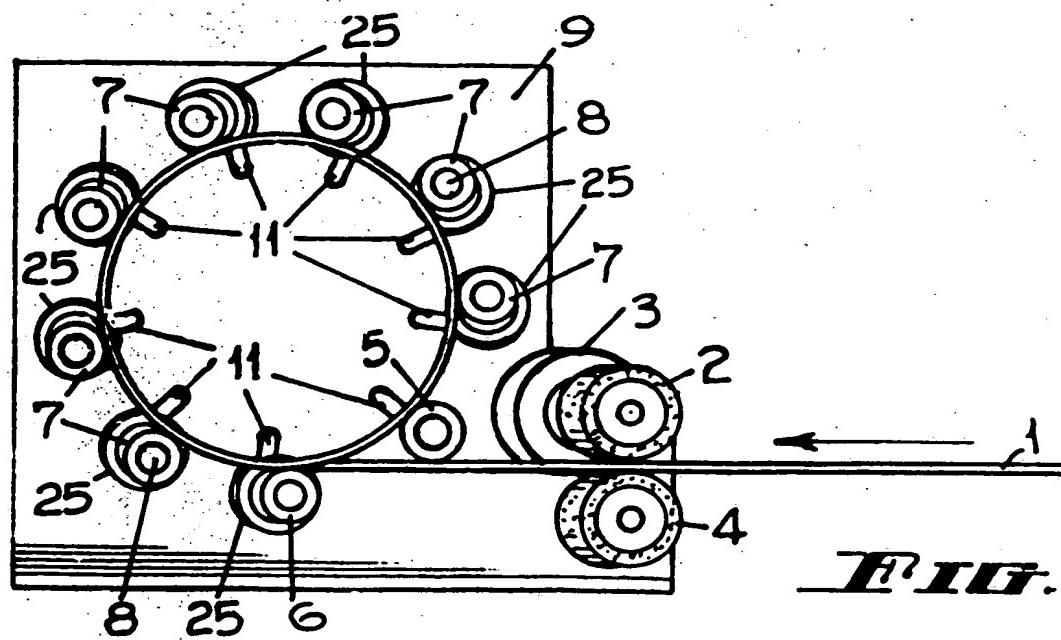
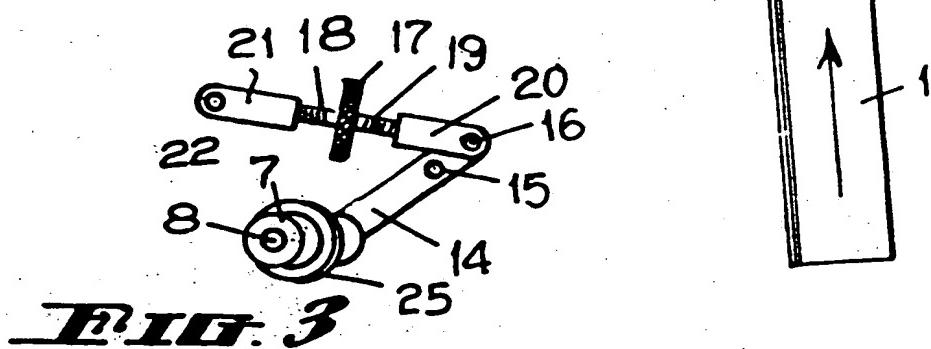
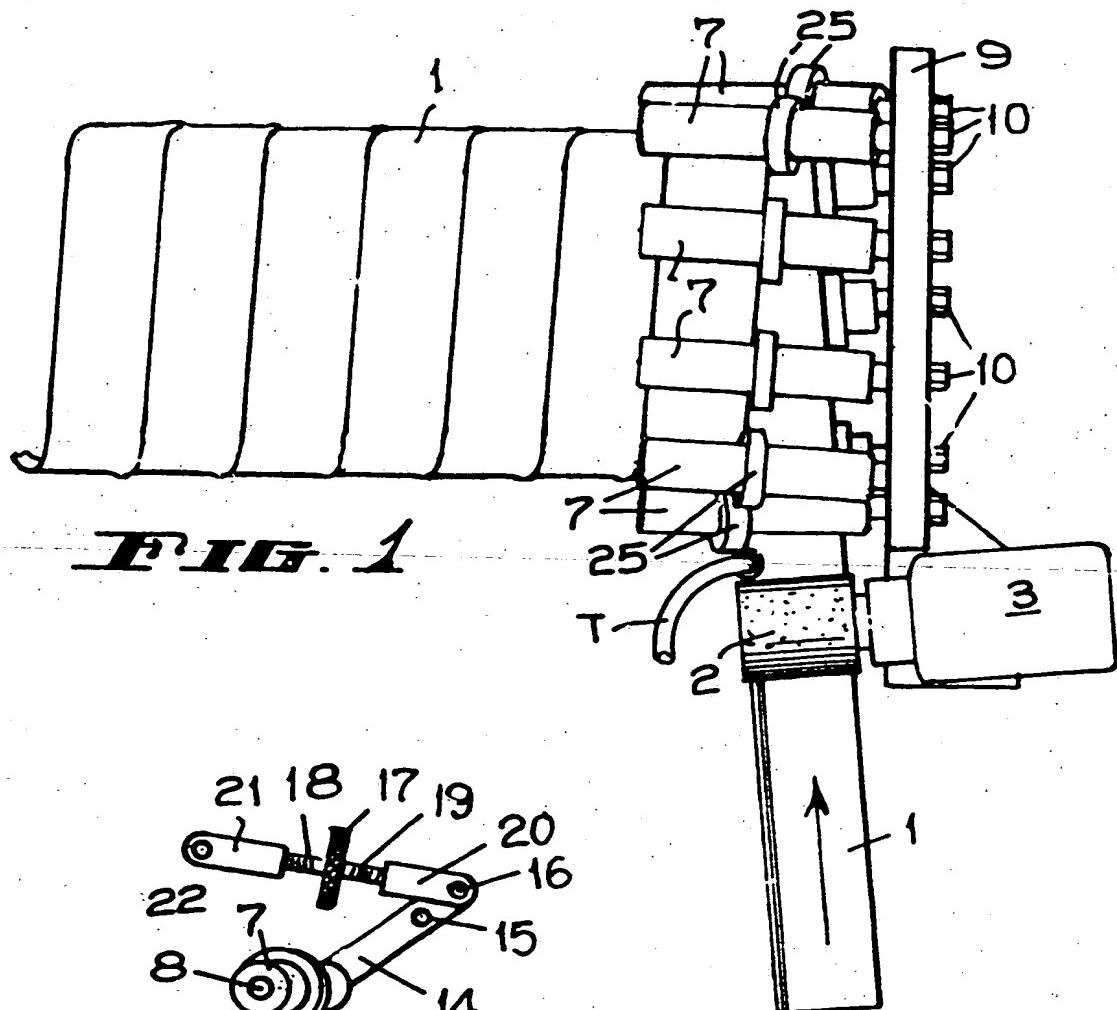
25. A machine according to any one of claims 13 to 24 wherein the said joining roller and the said drive roller or drive rollers are mounted on shafts carried on bearings in the said support frame, and
5 wherein the said support frame is mounted to be rotatable about the axis of the helical tube being formed.

26. A machine for forming tubes from strip substantially as hereinbefore described with reference to the accompanying drawings.

Dated this 6th day of April 1983

RIB LOC HONG KONG LIMITED
By their Patent Attorneys
COLLISON & CO.





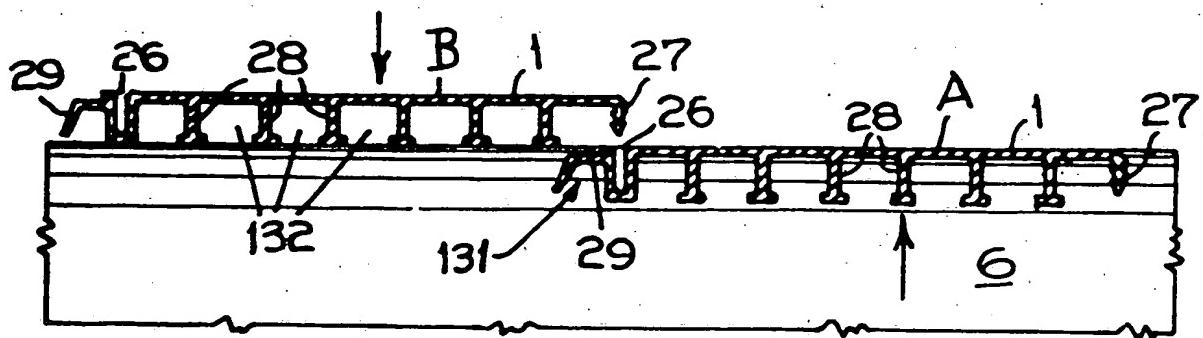


FIG. 4

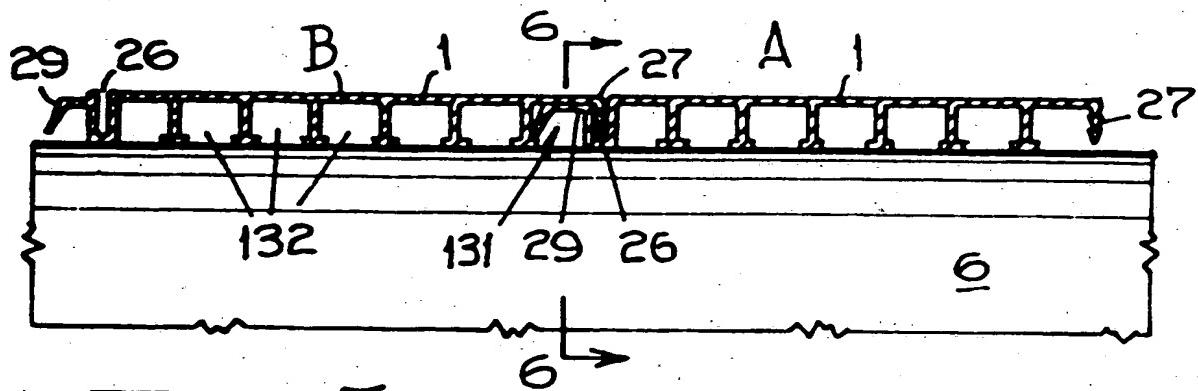


FIG. 5

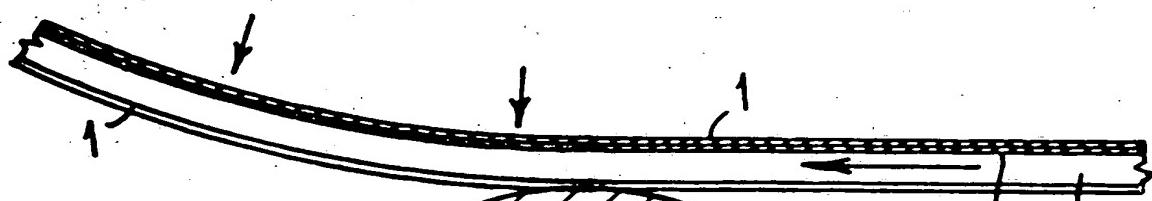
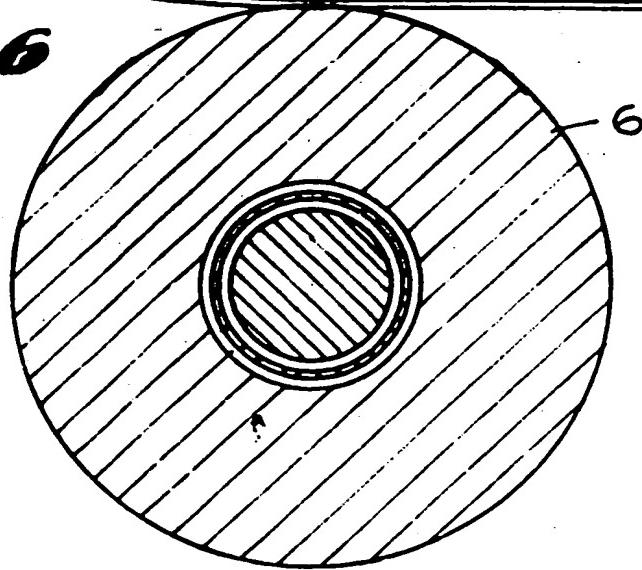


FIG. 6



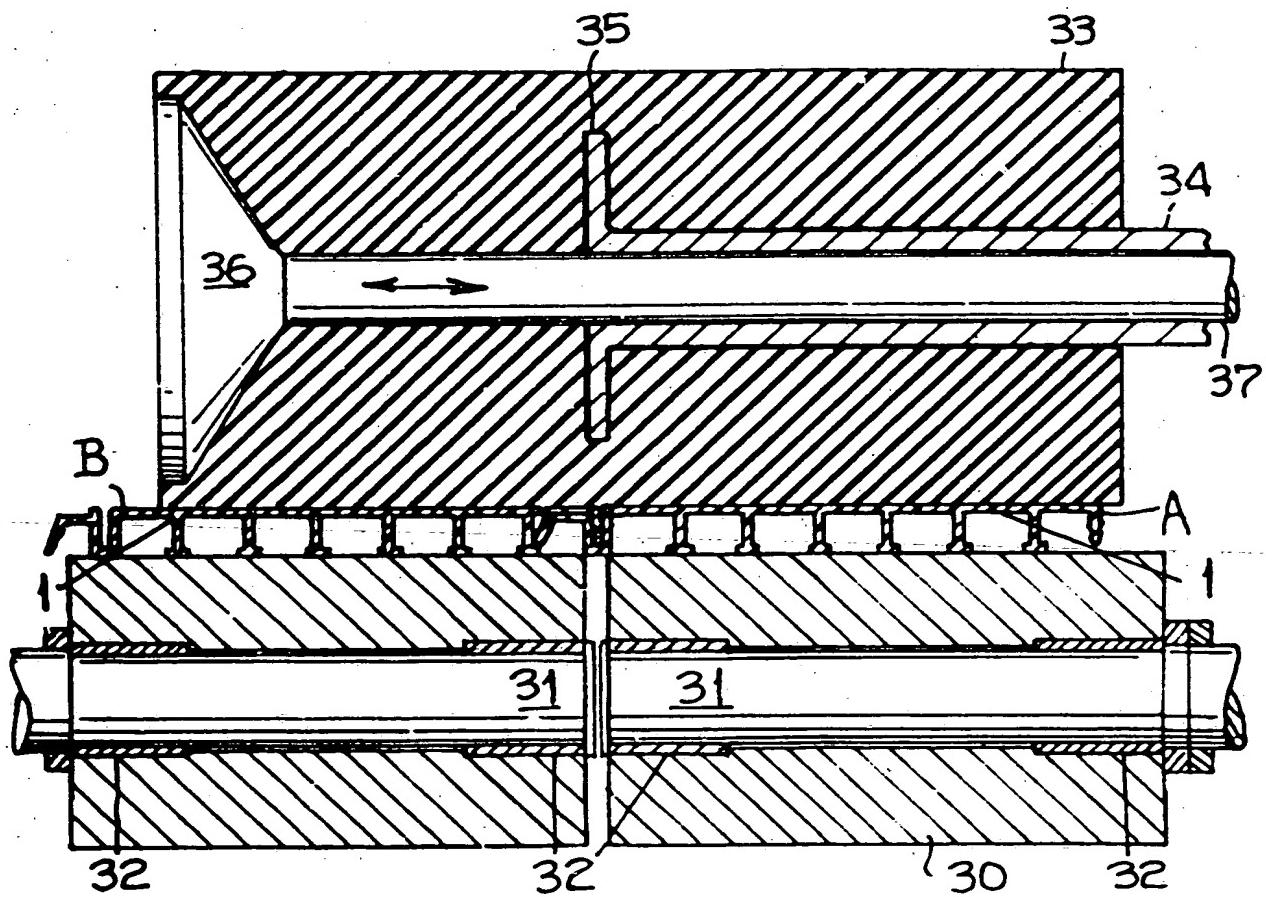


FIG. 7

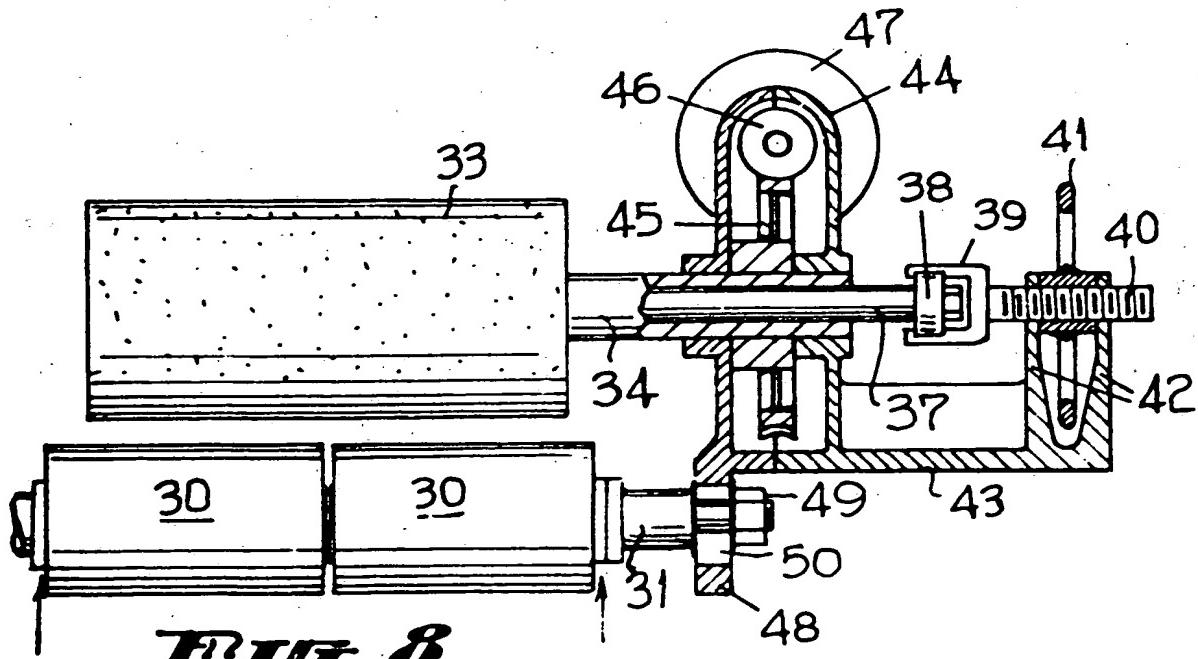


FIG. 8

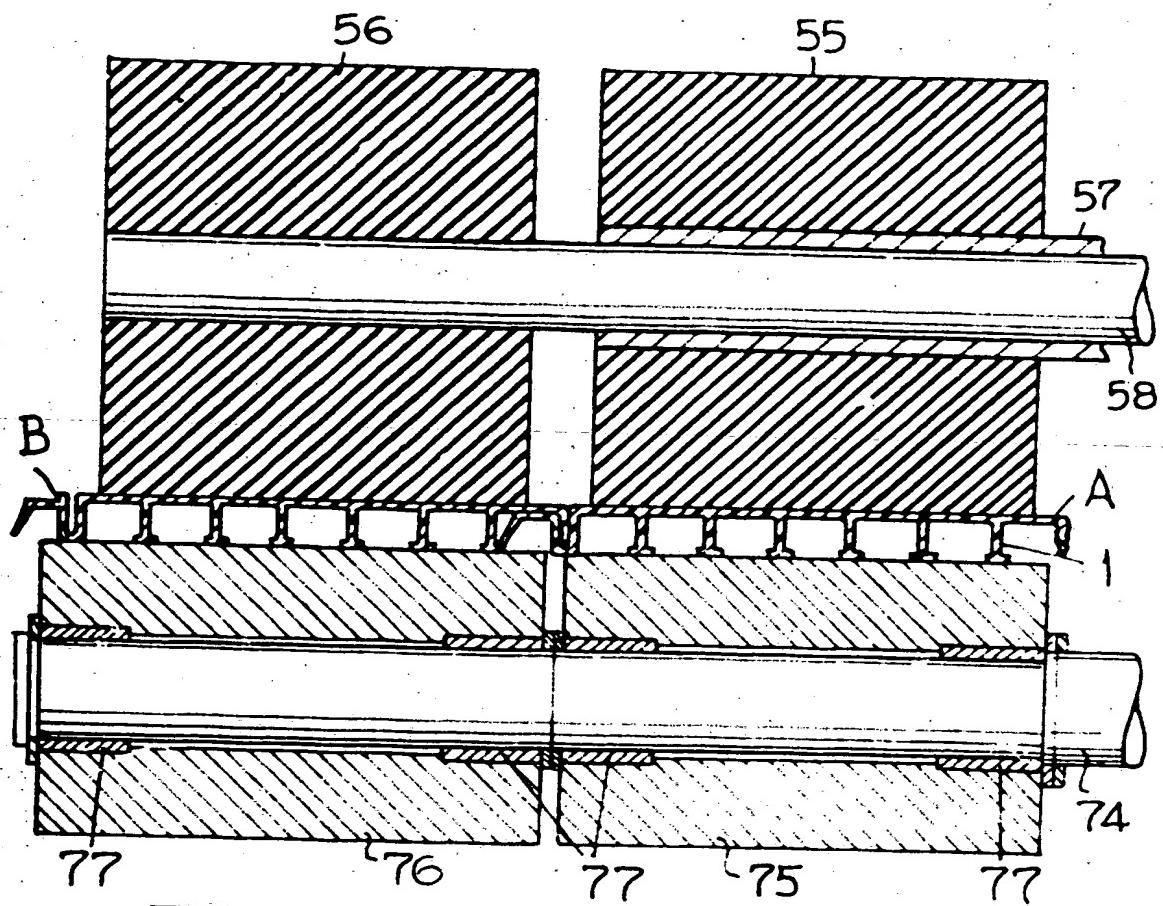


FIG. 9

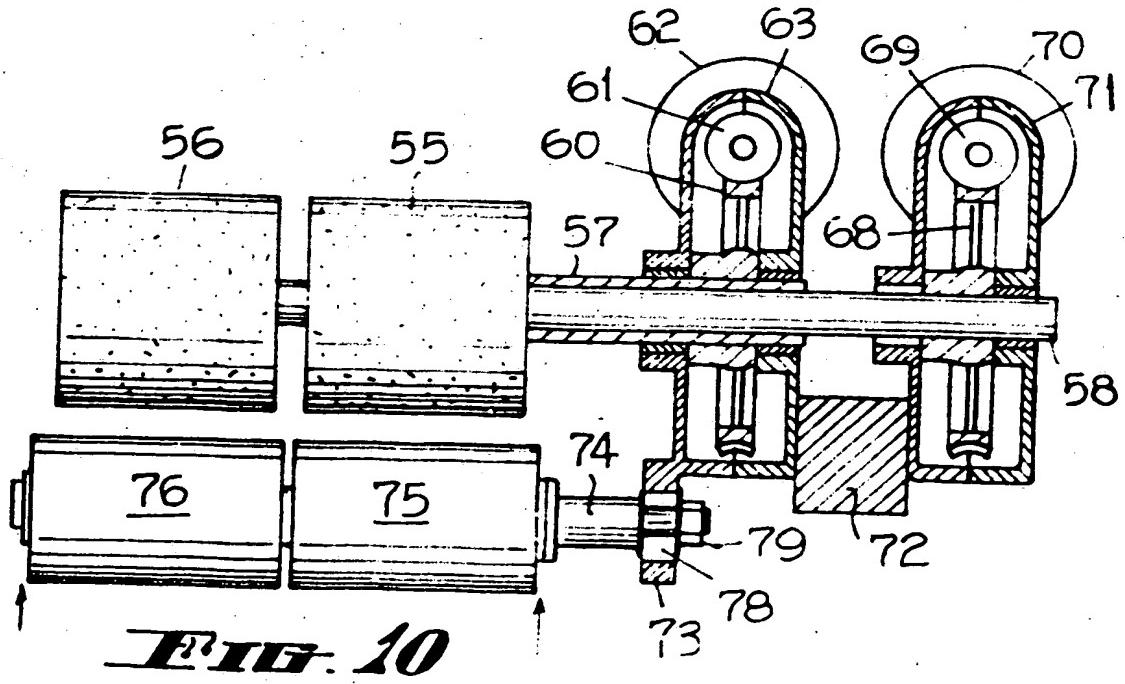


FIG. 10

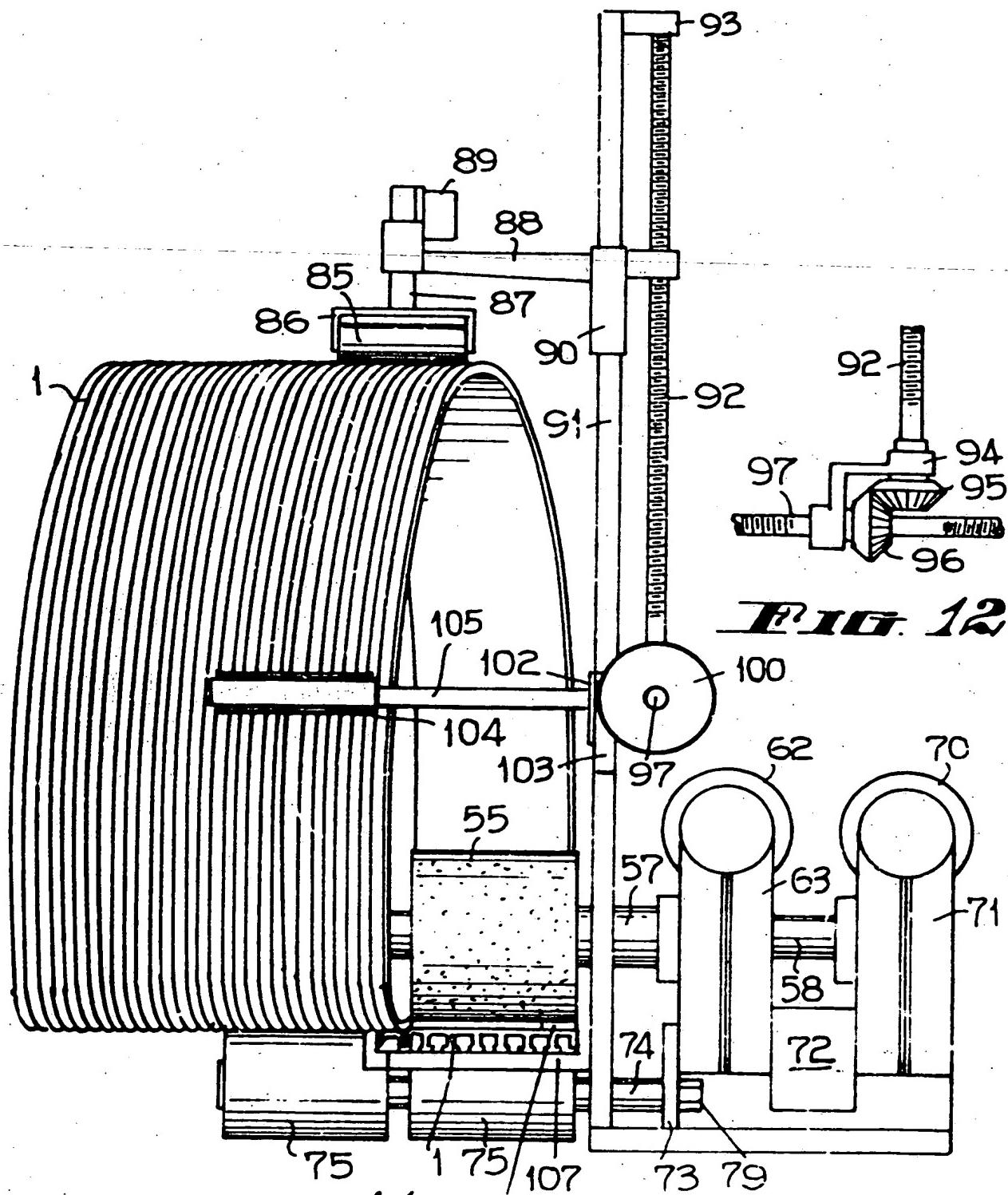
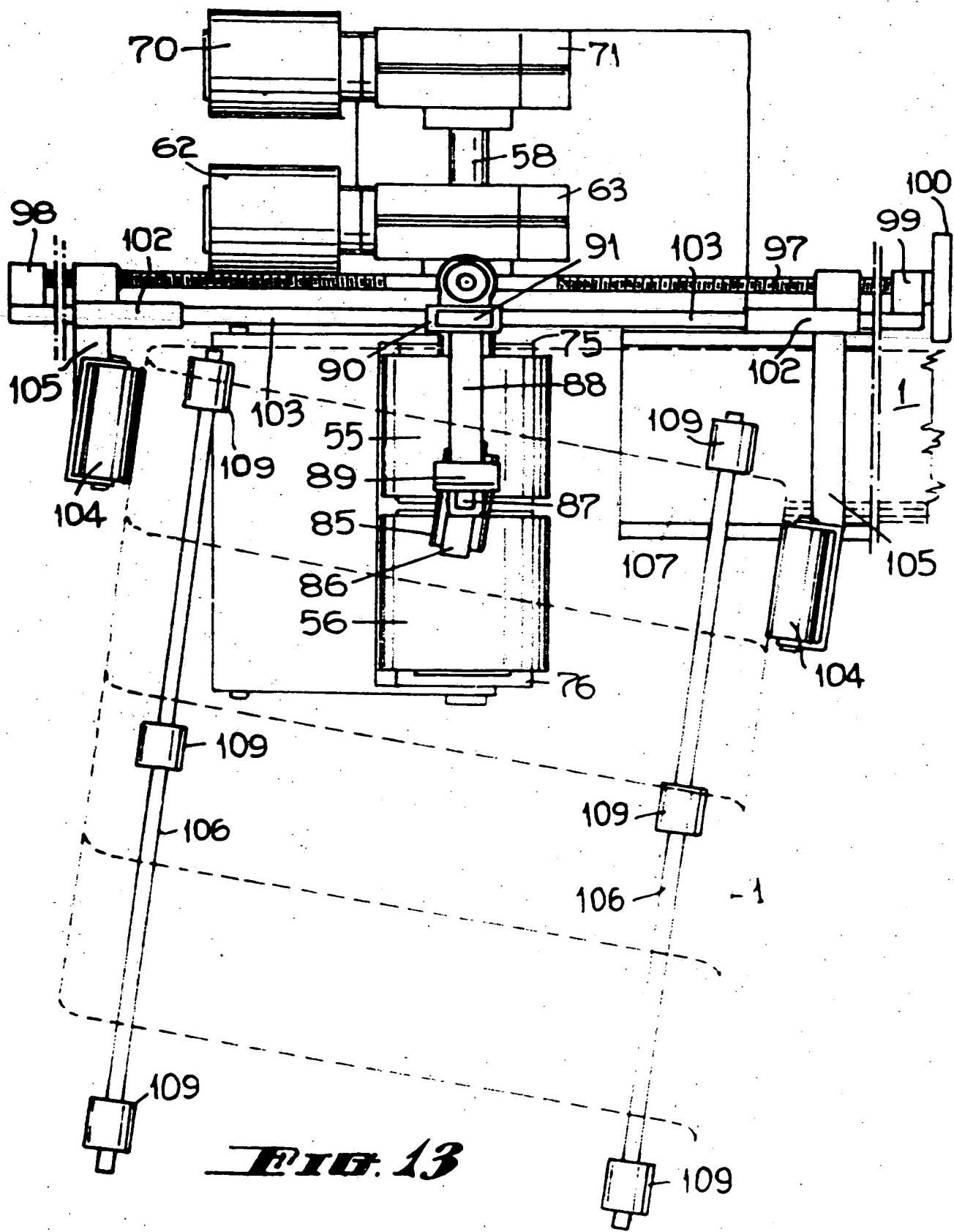


FIG. 12



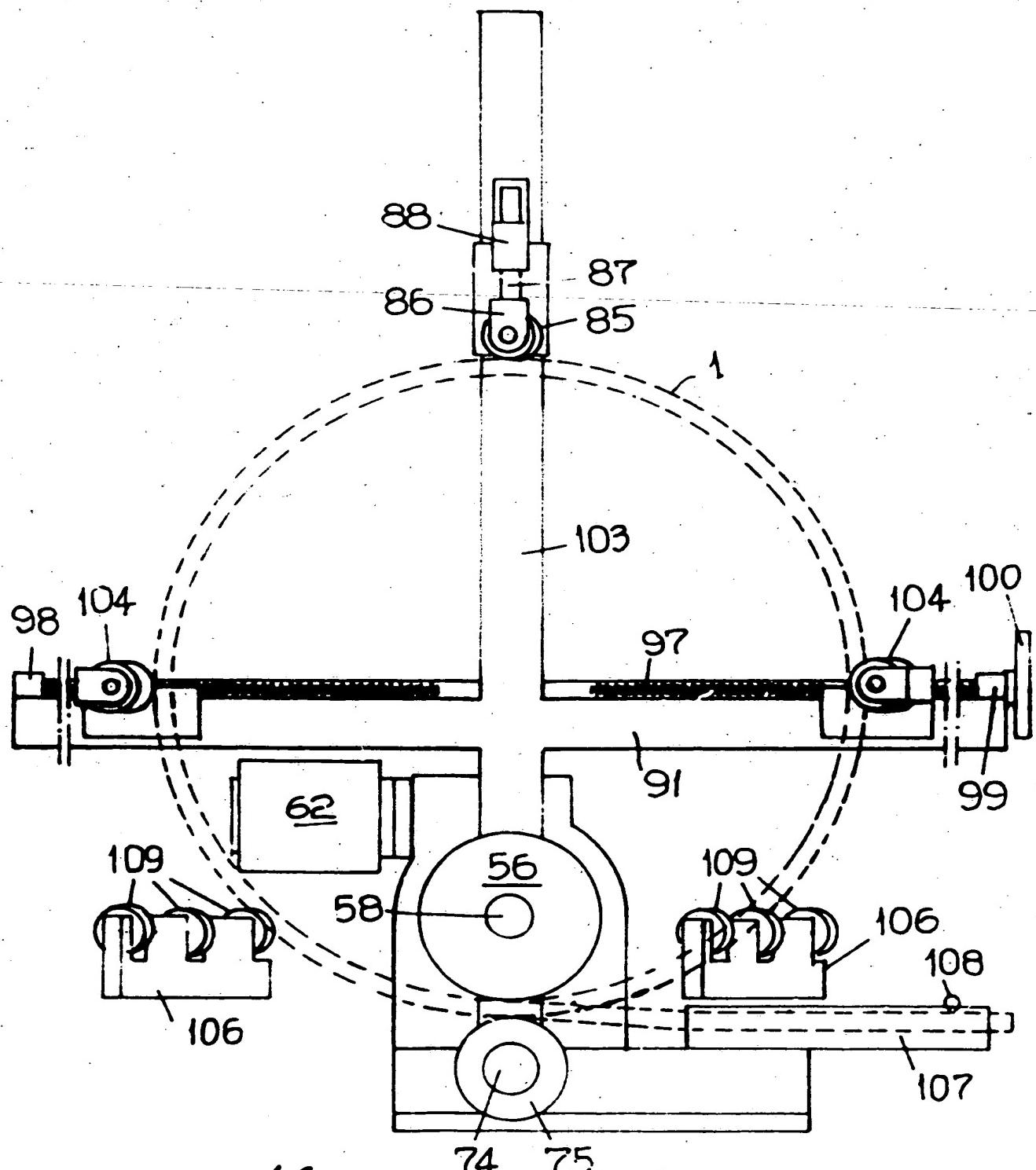


FIG. 14

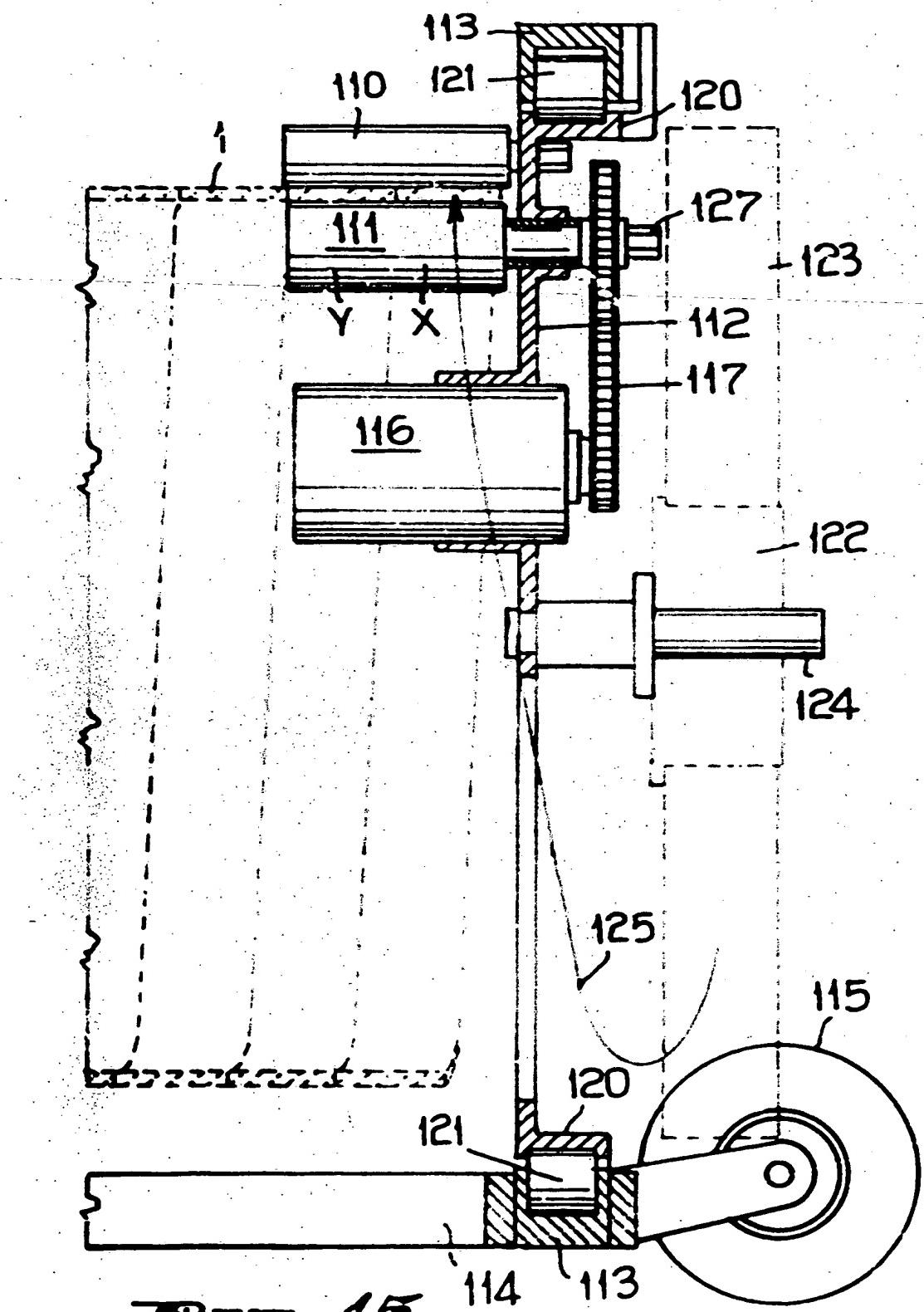


FIG. 15